

PipingSolutions, Inc.

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ANALYSIS OF STRESSES IN PRESSURE VESSEL AND TANK SHELLS AT NOZZLE AND CLIP CONNECTIONS RESULTING FROM EXTERNAL LOADS

WERCO 107/297 is a comprehensive software package for calculating stresses in shells in accordance with the guidelines set forth in the <u>W</u>elding <u>R</u>esearch <u>Co</u>uncil Bulletins 107 & 297. The program eliminates the need for hours of tedious hand calculations and manual cross-referencing and dramatically reduces the possibility of errors.



WERCO 107/297 APPLICATIONS

- Provides an effective tool for evaluating the ability of a nozzle to withstand the forces and moments applied by piping
- Provides an effective tool for evaluating the ability of a clip to withstand the force and moment loads applied by piping, structural members and equipment
- Provides an effective tool for evaluating the ability of a lifting lug to withstand the forces and moments applied to it at lift
- Provides technical compliance and documentation for regulatory authorities
- Enables an engineer/designer to determine the reinforcement required in spherical and cylindrical shell at the intersection of a nozzle, a clip or a lifting lug.

GENERAL FEATURES

- Easy to use Data Entry Dialog screens
- Easy to understand HELP screens
- Extensive Data Checking
- Imperial and International units
- Support provided by experienced engineers.

WERCO 107/297 CAPABILITIES

- Calculates stresses at eight points on the shell at the nozzle, clip or lug intersection
- Automatically compares calculated maximum combined stress intensity with the allowable stress value – automatically increases reinforcing pad thickness until the calculated stress value is found to be less than the allowed stress
- Determines worst combination of positive and negative loads and then designs a reinforcing pad so that the calculated stress is less than the allowed stress.



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WERCO 107/297 INPUT

- Shell Geometry Shape (cylindrical or spherical)
- Attachment Geometry shape (round, square or rectangular – solid or hollow)
- Attachment Orientation to shell
- Loads (forces and moments) X, Y & Z
- Internal Pressure acting on shell
- Joint efficiency, if applicable
- Maximum Allowable Stress Intensity
- Maximum Reinforcing Pad Thickness.

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WERCO 107/297 OUTPUT

- Input Echo with Error and Warning messages
- Stress Report in WRC 107 Spreadsheet Format
- Lists Bulletin Factors used in the calculations
- Required Reinforcing Pad Thickness
- Sizes Minimum Wall Thickness for nozzles
- Calculates Nozzle re-pads, if required

DW18: 05/22/2000		COMPRE	REALISE =			
* PAXINUM STRESS INTENSITY OF	TION SET	ACTED **	octon -			
* MERCO DESIGN OPTION SELECTS	ED -	200000000				
ALLOWABLE STRESS INTENSITY	(PSI)		15000	100		
DESIGNED RAD THICKNESS (IN)		0.250				
OTPUT DATA (ALL STRESSES)	ARE GIVEN	IN IPSI	ý	1		
			LUCAT	ION		
PIRCUMPERENTIAL TYPE DI STRESSES-	OF N	AU	AL	80	PL	
ELEPTERATE	(FY)	-595.	-595.	-595.	-595.	
ESND1043	(FY)	-2027.	2027.	-2027.	2027.	
MEMBRASIE	(NZ)	D	α.	D,	D.	
BENDING	(M3)	0.	0,	D.	0.	
RIPDRAME	(MIC)	2201.	2201.	-2201.	-2201.	
PENDING	(104)	6440.	-8440.	-644D.	6440.	
MEMBRANE STRESS	8E8	1606.	1606.	-2796.	-2796.	
BENDING STREES	KB	4412.	-4412.	-8467.	8467.	
TOTAL CIRCUMPERENTIAL I	TRESSES	6D19.	-2806.	-11263.	5671.	
PORGIJUDINAL SIBESSES- TIDE	DUE TO					
MEMBRASIE	(FY)	-402.	-462.	-402.	-402.	
E ENDING	(8.3.)	~2770.	2770.	-3770.	2770.	
PEPERARE	(M3)	D	Q.,	D.	0.	
PERID TRIG	(MZ)	n	а.	п.	D.	
MEMBRANE	CMX1	615-	615.	-615.	-615-	
PRINDING	(100)	10155.	-10155.	-30155.	10155.	
NENSRAME FTRES	and a	133.	133.	-10PE.	-1098.	
BENDING STRESS	88	7386.	-7386,	-12985.	12925-	
TOTAL LONGITUDINAL SIMI	ESSES	7518.	-7253.	~14023,	11828.	
HEAR STRESSES DUE TO						
TORSIONAL NOMES	(YN) TI	1079.	1079.	1079.	1079.	
SHEAR LOAD (F)	0	-692.	- 692.	692.	492.	
BHEAR LOAD (F)	\$2 (1	0.	0.	0.	0.	
TOTAL SHEAR STRESSES		387.	387.	1771.	1771.	
DOWNTHING ORDINAL TRANSIT	STRU	7612	7296	14000	12201	

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